

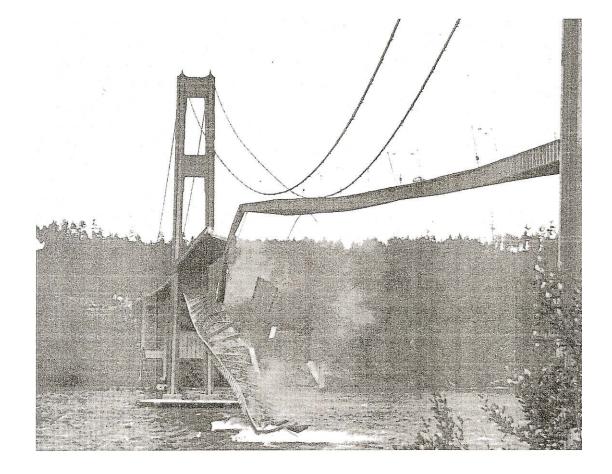
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Fatigue & Fracture Analysis "On the Fly"

Edward F. Punch Punch Software Solutions

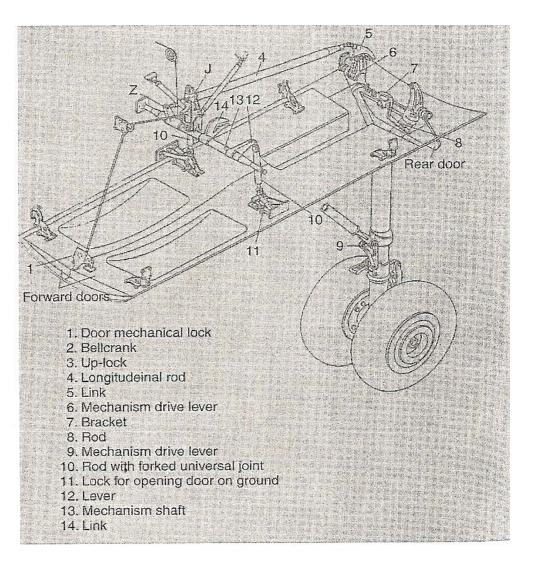


Telephone call. It is for you.....



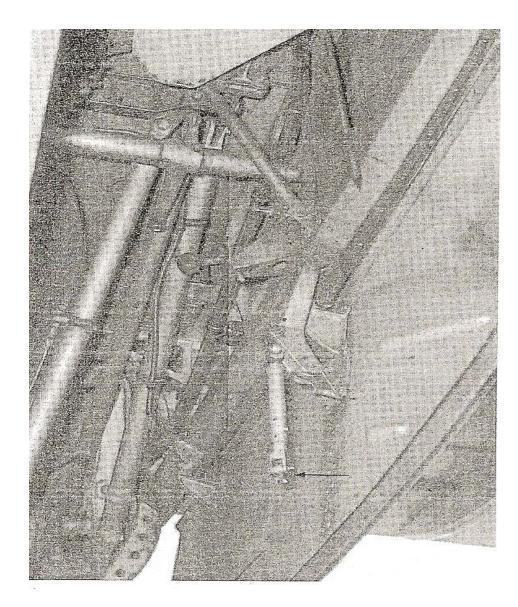






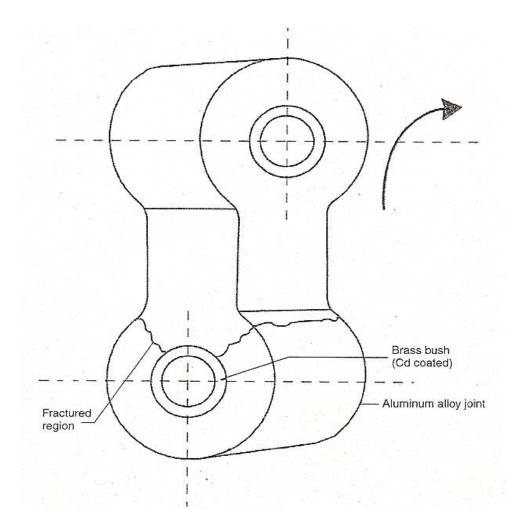






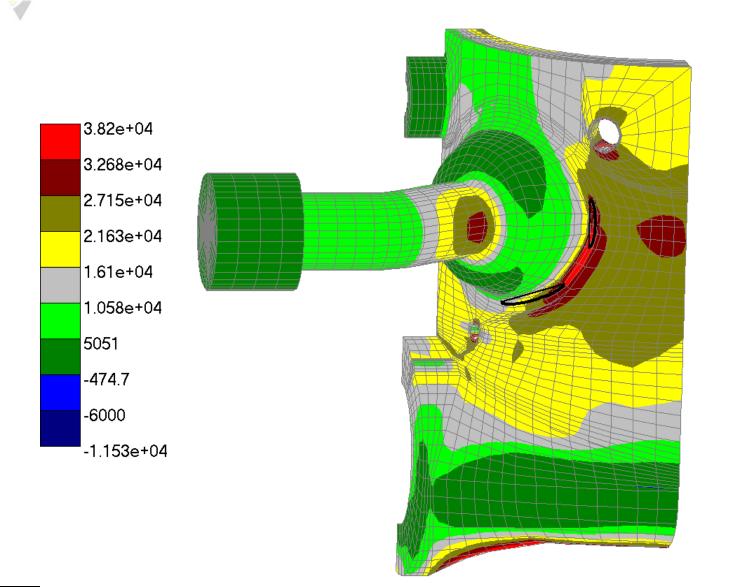








FEA + Superposition

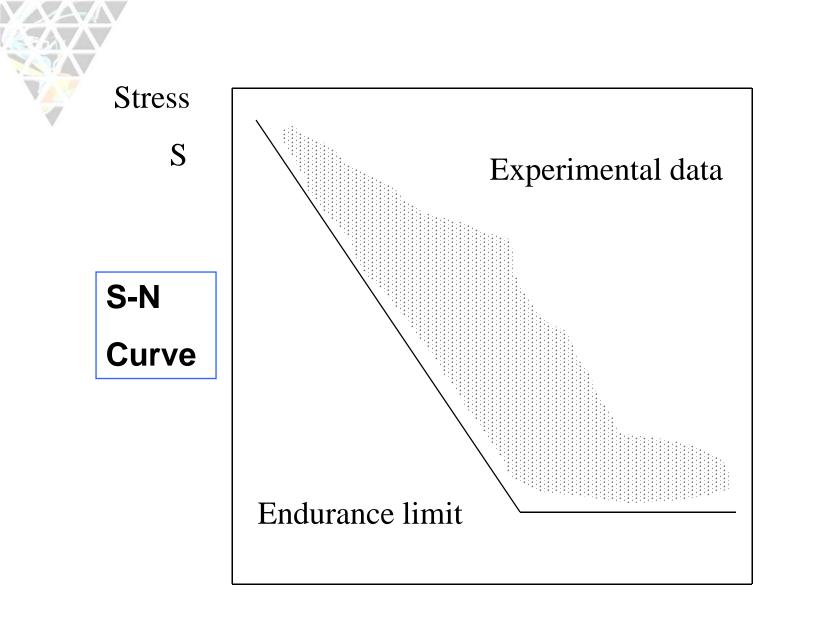




Fatigue design methods:

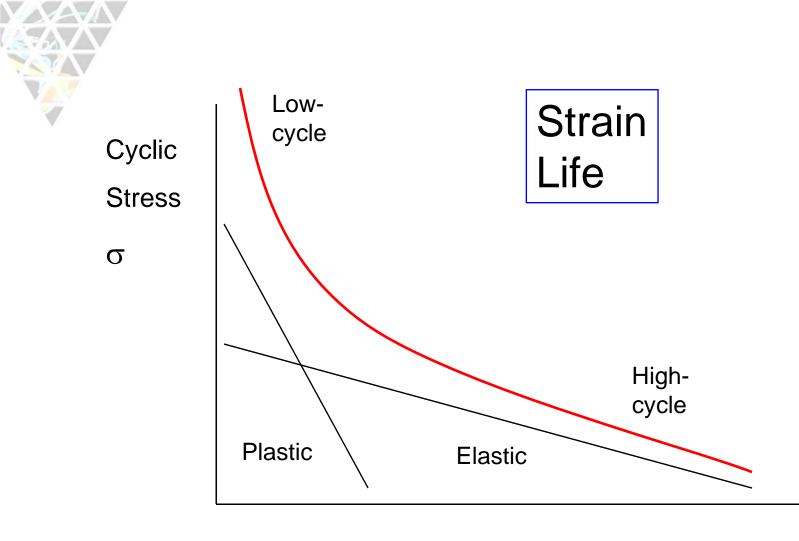
- Wohler S-N curve (stress proxy)
- Strain-life approach (strain proxy)
- Continuum damage theory
- Fracture mechanics





Cycles N





Lifecycles N



Drawbacks of Strain-life

- Crack is not modeled
- Low-cycle fatigue only
- Difficult to fit experimental data
- Specific to residual stress & surface treatments
- "Push-button" strains are computed from coarse tetrahedral FE meshes → Failures



Continuum damage theory vs. elasto-plasticity

- Elastic modulus
- Poisson ratio
- Yield stress
- Ultimate stress
- Critical damage D_c
- Asympt. fatigue σ_f
- S & s → Dissipative potential function

- Elastic modulus
- Poisson ratio
- Yield stress
- Ultimate stress



Fracture mechanics approaches

- Model the crack and crack growth
- Use tables of standard K-factors (e.g. NASCRAC)
- Mesh the crack front (e.g. ABAQUS)
- Superimpose cracks on *existing* finite element meshes → Alternating finite element method (AFEM)



Subsurface defects by AFEM

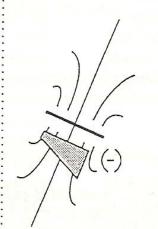
+

Uncracked stress distribution (+)

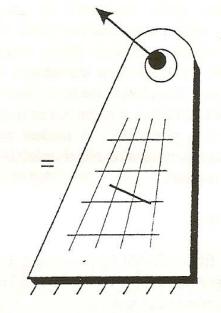
> Uncrocked FEM stress distribution (tensile)

domain

Infinite



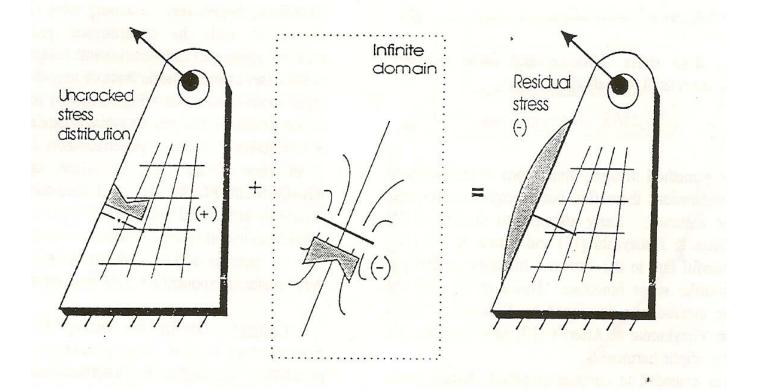
Crack stress field (compressive) in infinite domain



Stress-free surface (ie. crack) in component

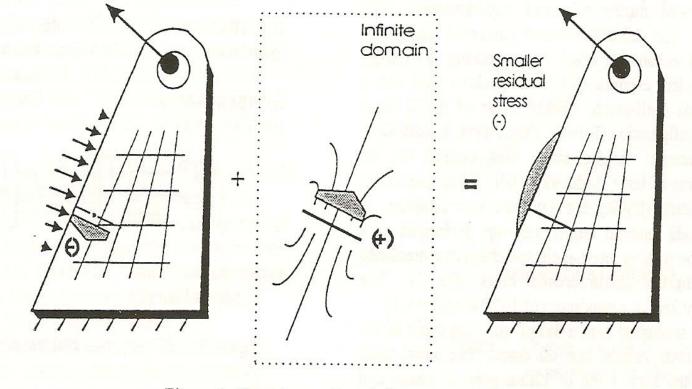


Surface defects





Iterative corrections





Modified Forman crack growth $\frac{da}{dN} = \frac{C.(\Delta K)^{n}.(1-R)^{m}.[\Delta K - \Delta Kth]^{p}}{[(1-R).K_{Ic} - \Delta K]^{q}}$

Lower bound: Δ Kth (threshold K-factor)

Upper bound: KIc (fracture toughness)

Different crack growth rates in base & carburized metal



Effect of Residual Stress Treatment

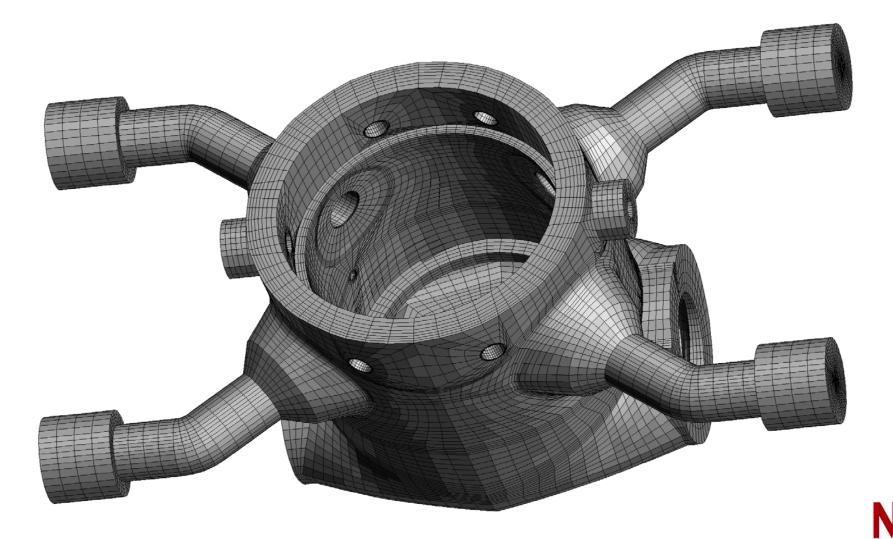
- Stress ratio = R
 - = Min/Max stress
 - = (Cyclic min + residual stress)

(Cyclic max + residual stress)

• where the exact process RS is used.

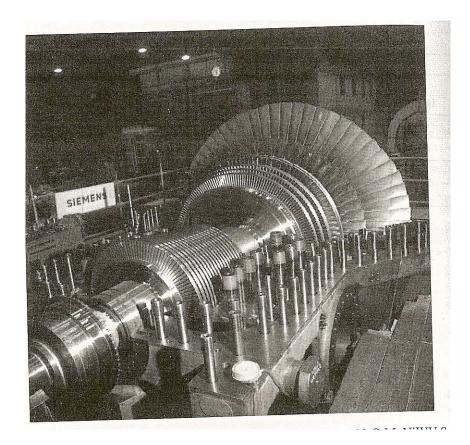


High-pressure turbine casing



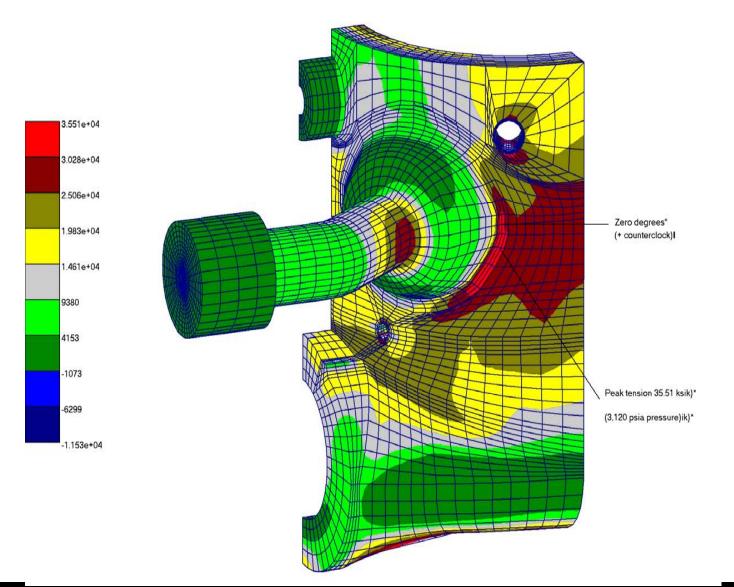
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HP, IP & LP turbine blades





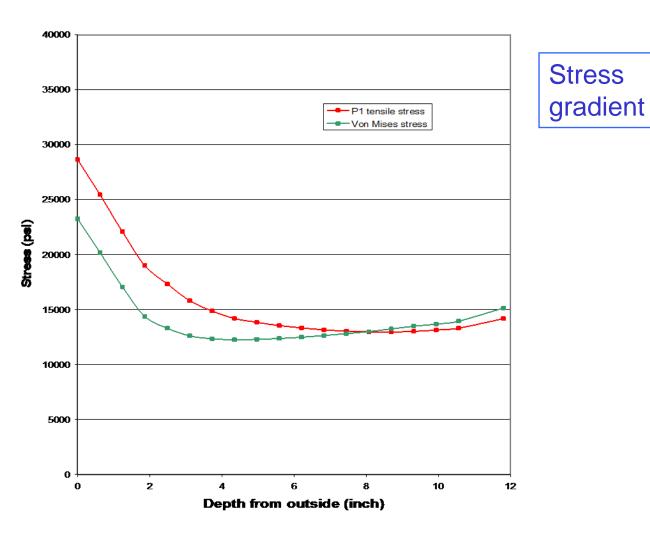
Principal tensile stress



Typical surface cracking



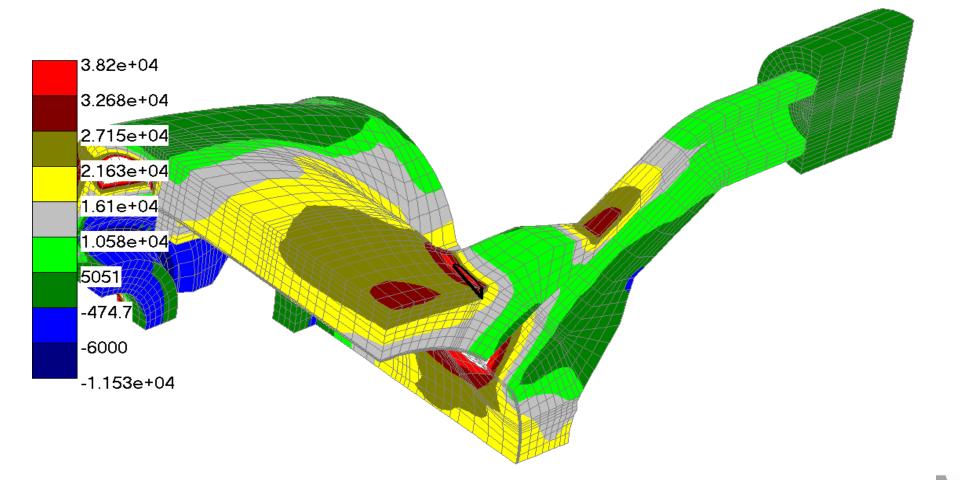


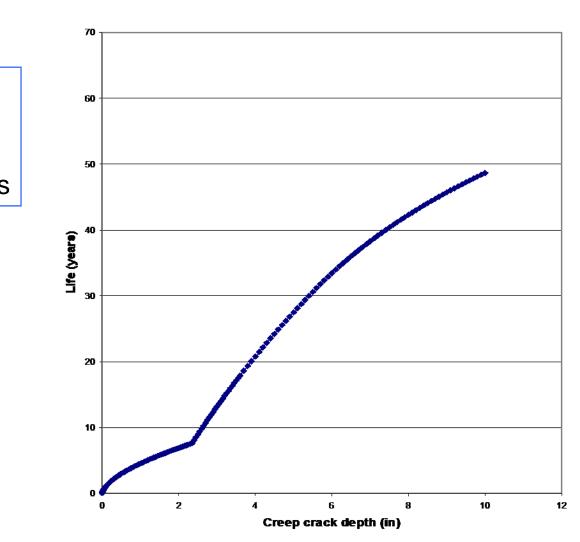






Superimposed surface crack

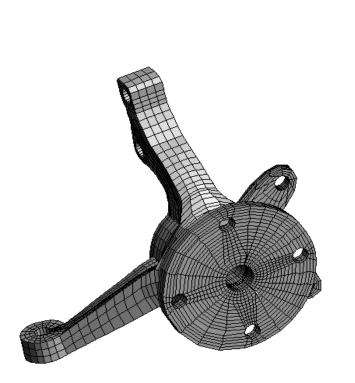


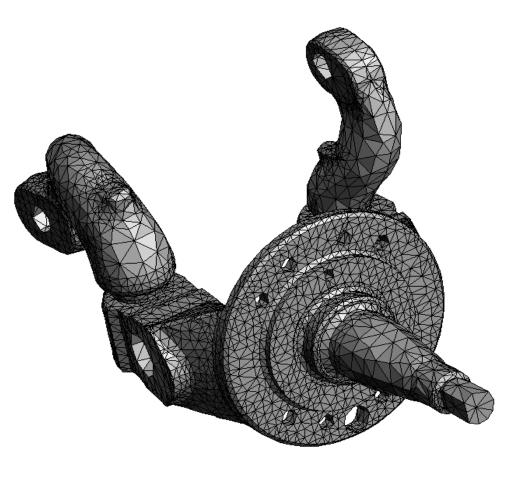


Crack depth vs lifecycles

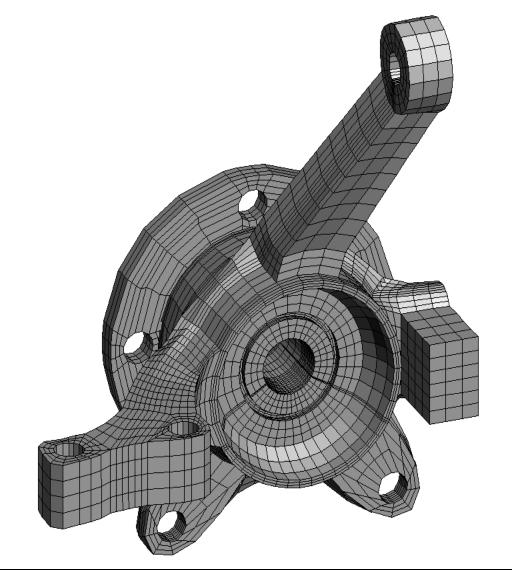


Steering knuckles





Midsize sedan knuckle





Measured wheel loads time history

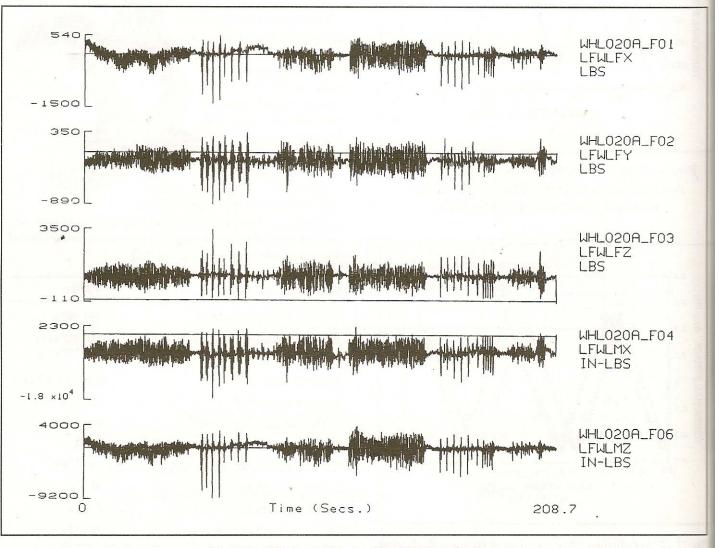
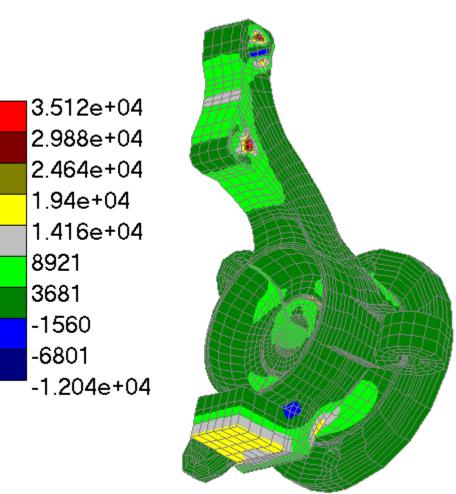


Fig. 11 - Measured wheel loads time history

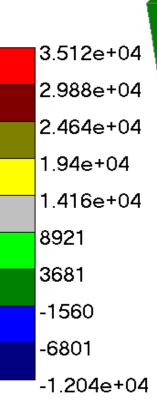
and Simulation

Principal tensile stress





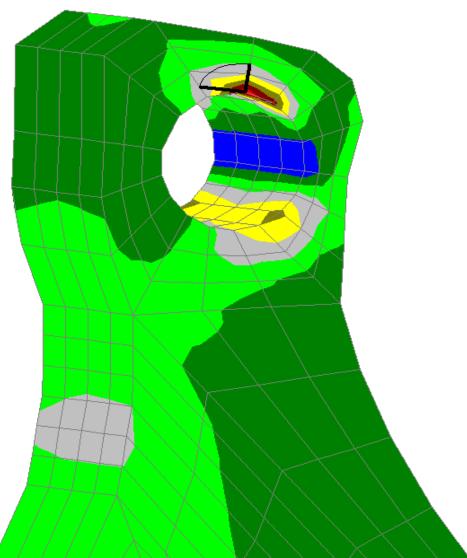
Corner crack at bolt hole

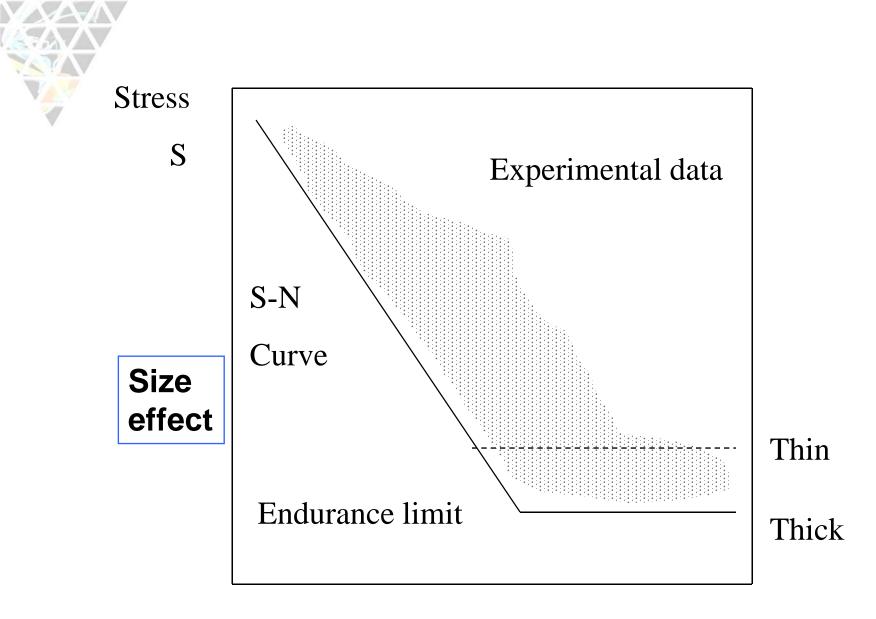




Corner crack growth

3.512e+04 2.988e+04 2.464e+04 1.94e+04 1.416e+04 8921 3681 -1560 -6801 -1.204e+04





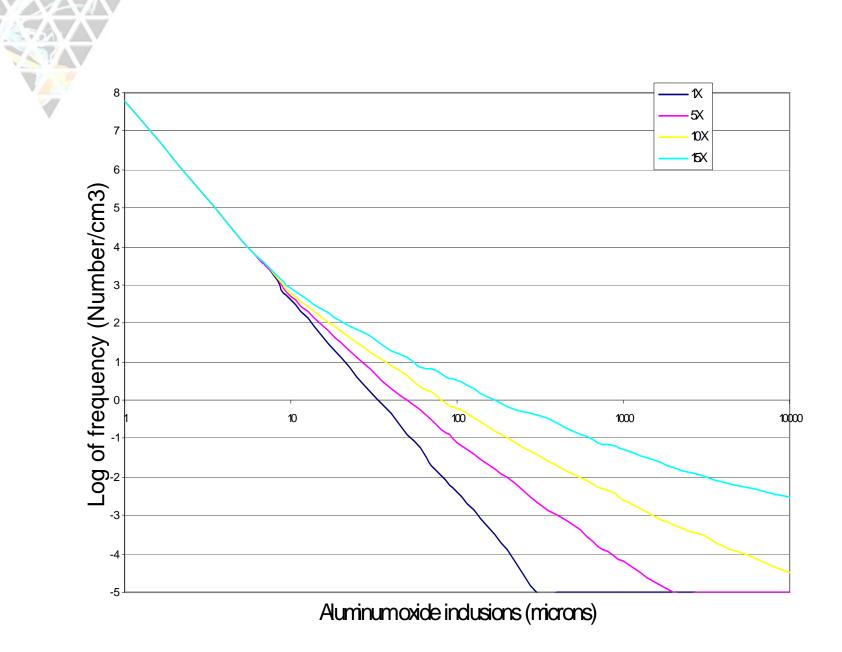
Cycles N



Fatigue specimen size effect?

- Stress life & strain life theories can not explain it
 - Explanation by stochastic fracture mechanics
 - Steelmakers data on inclusion & defect sizes
 - Large specimens -> more material -> more defects -> more variability -> shorter life









- Stochastic fracture mechanics →9.115% size effect
- Experimental tests \rightarrow 10% size effect
- 1000s of Monte Carlo crack growth simulations performed by fast accurate Alternating Finite Element Method (AFEM)

